

Experience with Hydro Generator Stator Noise due to Air Gap Harmonic Forcing

Thomas J Cunningham, P. E.

Fellow Engineer

ElectroMechanical Engineering Associates, Inc.

Pittsburgh, PA USA

Problem:

- *Nice, new, continuously stacked Stator*
- *New coils & winding configuration*
- *Often, higher power output*

- ***And it screams like a Banshee!***

What Happened?

Slot passing frequency harmonics

- *Inherent to fractional slot windings*
- *Not always audible*
- *Frequencies at multiples of 2x synchronous speed*
 - *Typically 8x, 10x, 12x, 14x*
- *Not just annoying – often require **full-time hearing protection***
 - *Author encountered 117 dB in generator barrel*
(between “rock band” and “Police siren” on the charts!)

Second problem – 2x line frequency pulsations

- *Same scenario – new core/winding*
- *Also inherent to fractional slot windings*
- *Not always audible*
- *Result of multiple waves at 2x line frequency*
 - *Recent case > 107 dB in generator vicinity*

OEM Generator Construction

- **Stators factory built – 3 or 4 pieces**
- **Shipped to site wound**
- **Final connections in field**
- **Frame joined – stator iron not**
- **Machines often 40 – 80 years old (or more!)**

Pressures and Opportunities

- Power needs increasing steadily

BUT...

- Coil Insulation Improved
 - More copper in slot
 - Higher temperatures allowed
- CFD improves turbine efficiencies

RESULT – UPDATES!

Some or all of these:

- Changes to # of Stator Slots
- Multi-turn > Single turn Roebel
- Segmented > Continuous Stator Core

And definitely these:

- Higher Field Current & Flux
- More Torque

Ok... So where's the problem?

(warning – math ahead)

Fractional Slot Windings



$$\frac{\text{\# of Stator Slots}}{\text{\# of Poles}} = (\text{not a whole number})$$

So what happens? (Examples easiest)

$$\frac{360 \text{ slots}}{56 \text{ poles}} = \frac{45 \text{ slots}}{7 \text{ poles}}$$

Pattern of 45 slots repeats over 7 poles ($6 \frac{3}{7}$ slots/pole)

Fractional Slot Windings



Slot passing (high frequency) harmonics:

$$\frac{360 \text{ slots}}{56 \text{ poles}} = \frac{45 \text{ slots}}{7 \text{ poles}} = 6\frac{3}{7}$$

Fraction – $3/7 < 1/2$ so 6th harmonic predominates:

$$6 \times 2 \times 60 \text{ Hz} = 720 \text{ Hz}$$

Fraction x no. of poles = waves:

$$\frac{3}{7} \times 56 = 24 \text{ waves}$$

Fractional Slot Windings



Electrical (2x Line Frequency) Pulsations:

$$\frac{360 \text{ slots}}{56 \text{ poles}} = \frac{45 \text{ slots}}{7 \text{ poles}}$$

$$\frac{\text{\# of poles}}{\text{poles in repeating pattern}} = \frac{56}{7} = 8 \text{ waves}$$

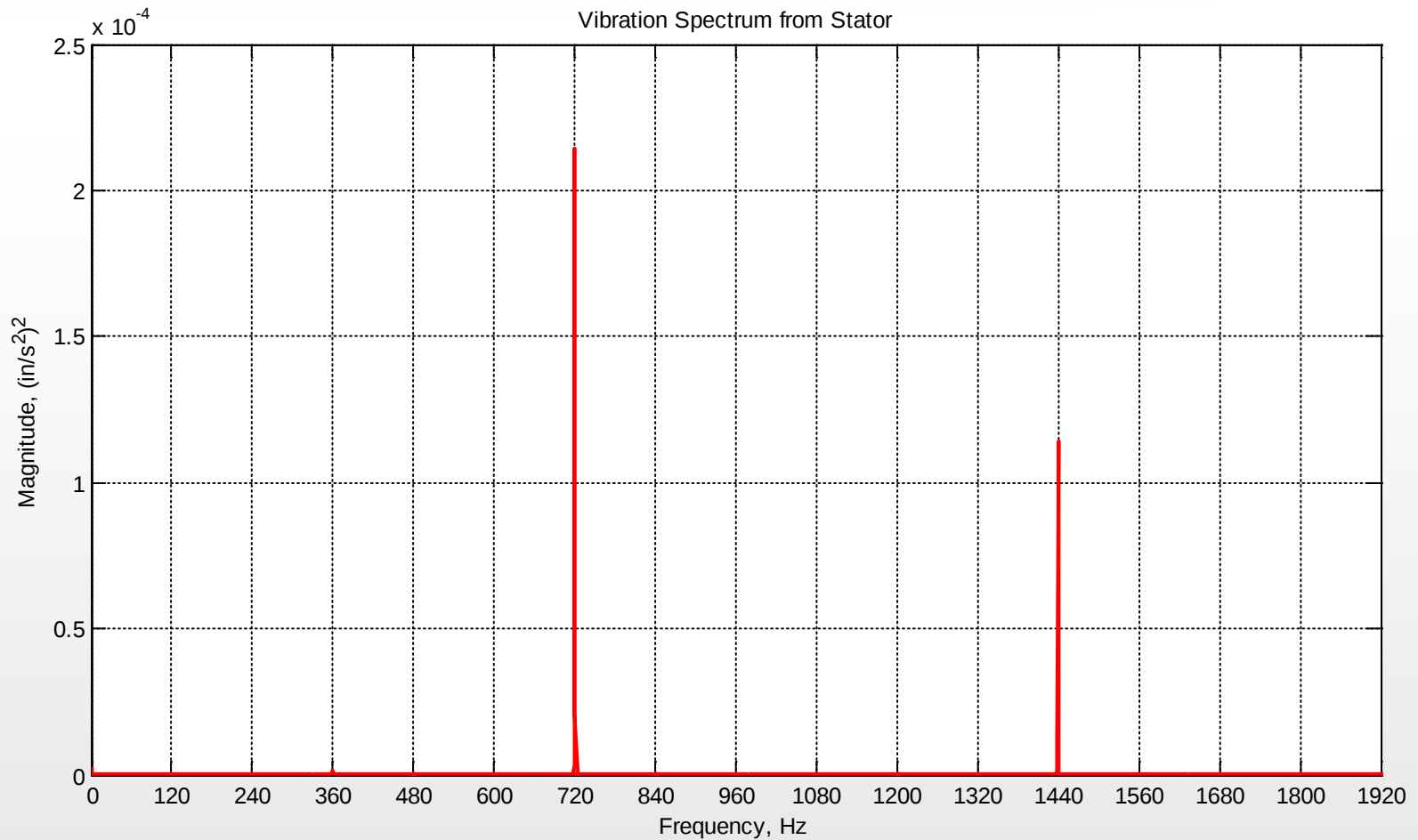
Both this and the 720 Hz AND a smaller 840 Hz wave exist

But here's what we heard:



This is machine "B" in the paper.

Vibration (Sound) Spectrum

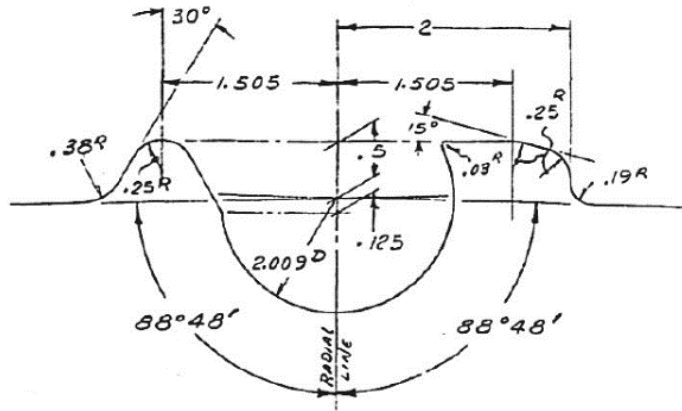


Stator Construction



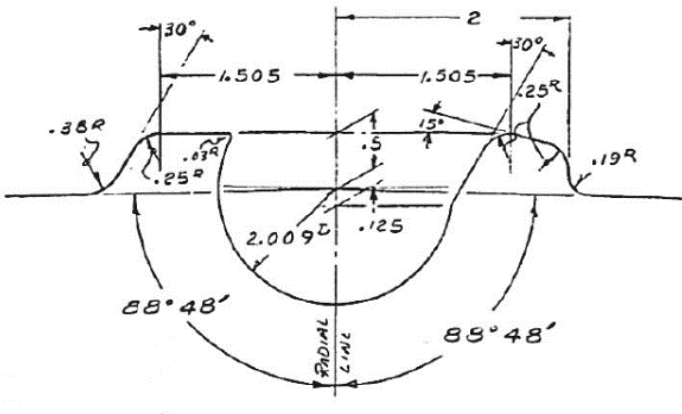
Powering Engineered Solutions

Lamination – Building Bolt Clearance

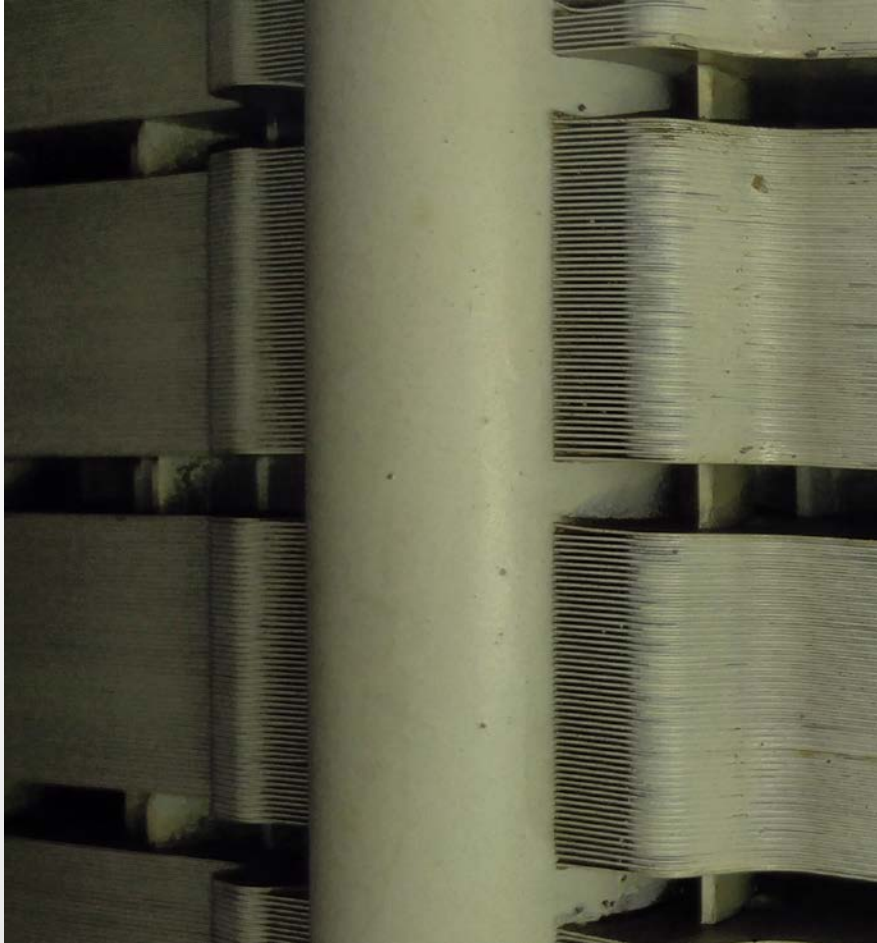


- Socket diameter – 2.009 in.
- Min. throat opening – 1.979 in.
- Building Bolt diameter – min 1.970"

Each lamination can only make point contact with the building bolt.

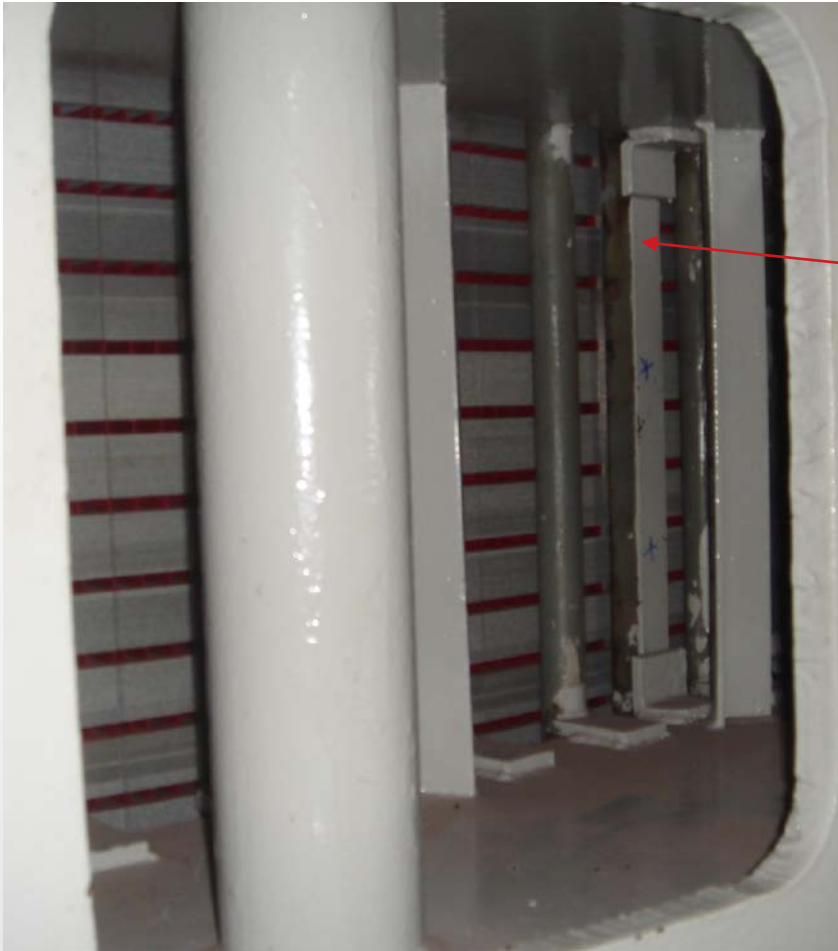


Building Bolt Contact



- Contact with building bolt is at best a point contact.
- Cannot prevent rotation of stator cross section.

Building Bolt & Frame Influence



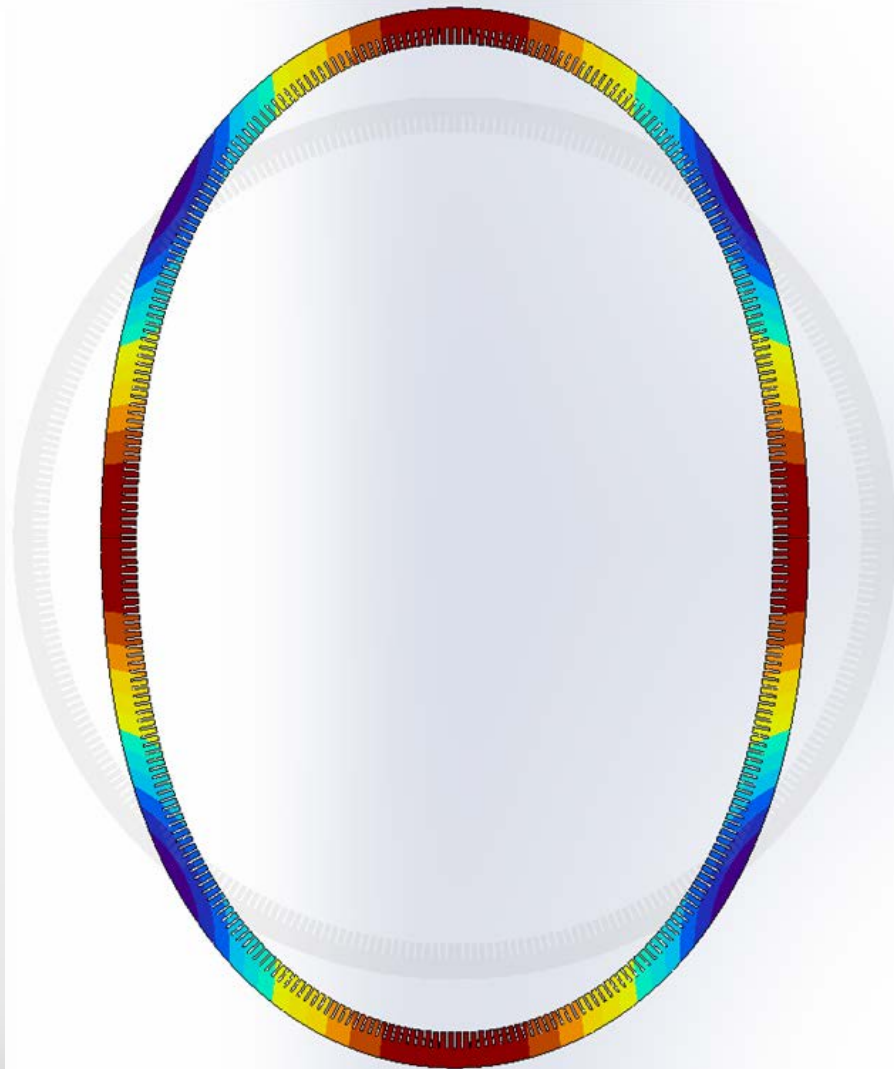
- Building Bolts or Keybars welded to frame rings
- Relatively soft between frame rings
- Do little to restrain core radial movement between Frame rings.

Natural Frequency of Core Iron

- ODS Tests clearly showed 24 waves at 720 Hz
- Resonance was painfully obvious
- “Thin Ring” Natural Frequency Calculation of core iron *alone*:
 $n = 24$ at 880 Hz



Stator Cores are DEFINITELY “Thin Rings”



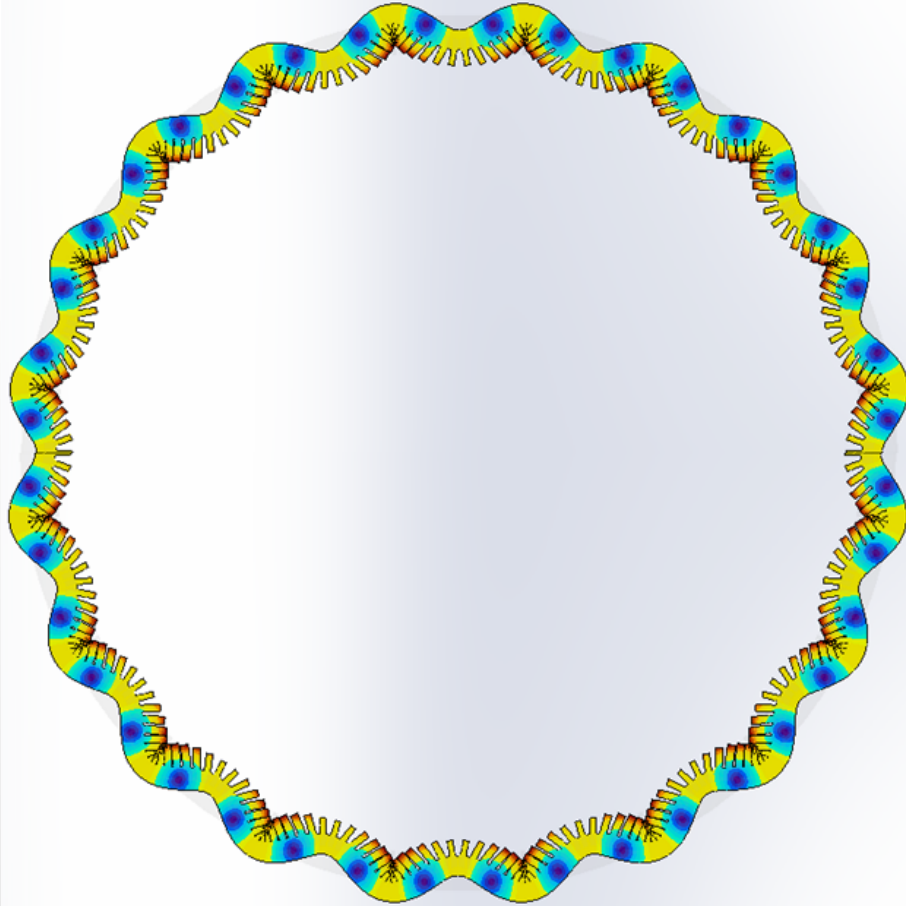
$$\frac{\text{Mean Radius}}{\text{Back Iron}} = 22.1$$

“Thin ring” vibration calculation first published by R. Hoppe, 1888

$$\omega_n = \frac{n(n^2 - 1)}{\sqrt{n^2 + 1}} \sqrt{\frac{EI}{\rho A R^4}}$$

Calculates two wave (n=2) at 5.81 Hz
FEA model at right - 5.56 Hz

Except when they aren't!



“Thin ring” vibration calculation first published by R. Hoppe, 1888

$$\omega_n = \frac{n(n^2 - 1)}{\sqrt{n^2 + 1}} \sqrt{\frac{EI}{\rho AR^4}}$$

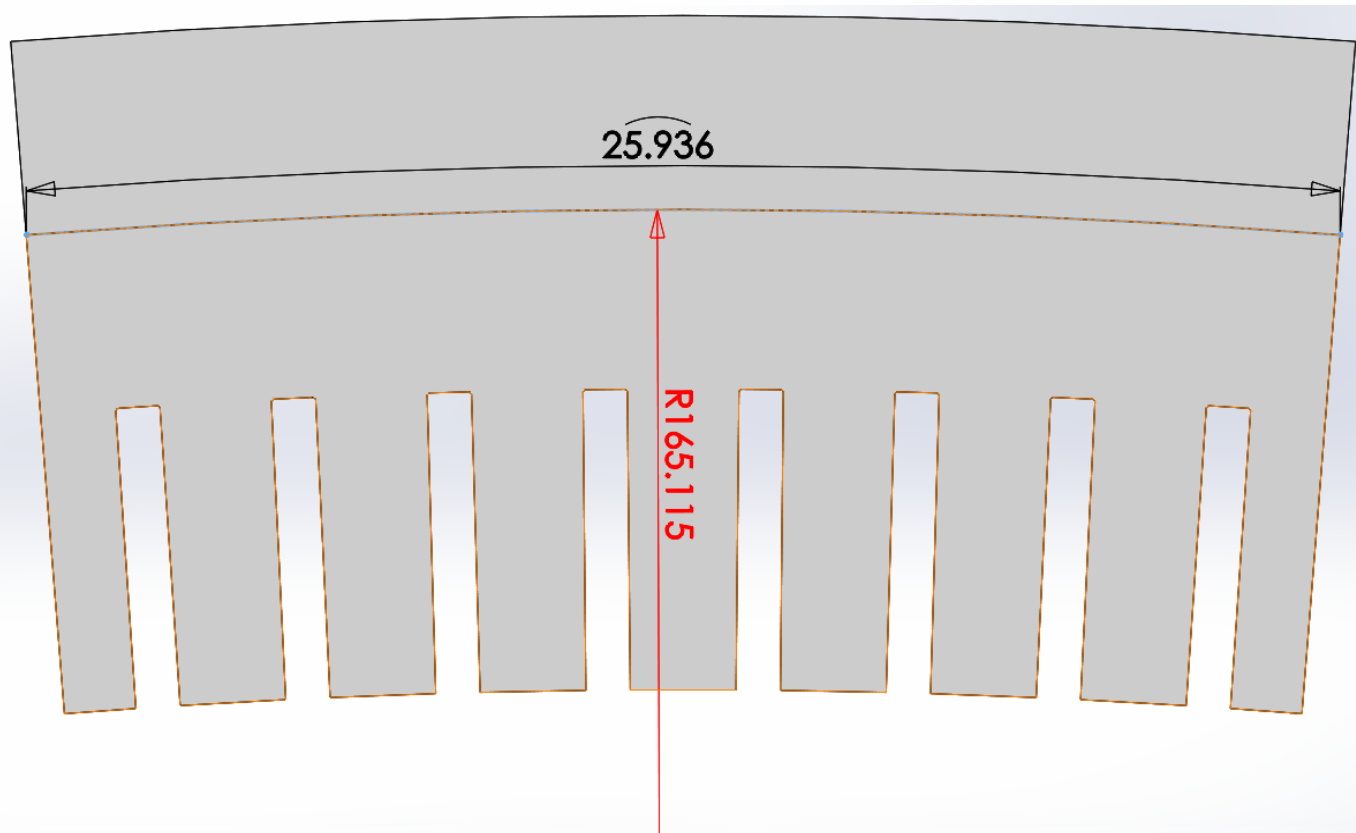
Calculates twenty wave (n=20) at 863 Hz
FEA model at right – **667 Hz**

$$n = 2 \quad \frac{5.81 \text{ Hz}}{5.56 \text{ Hz}} = 1.044$$

$$n = 20 \quad \frac{863 \text{ Hz}}{667 \text{ Hz}} = 1.294$$

Mass/stiffness ratio difference - 1.64!

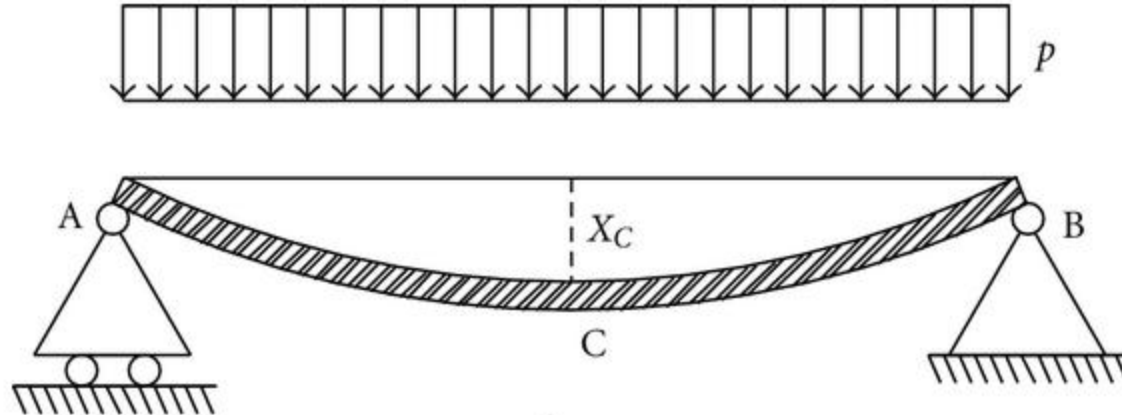
Except when they aren't!



This is one half-wavelength of the stator for the twenty wave mode
Euler "slender beam" theory $L/D > 10$
 L/D here is 3.47

Variances from “Thin Ring” formulation

Consider $\frac{1}{2}$ wavelength of stator bending as simply supported beam:



- Wavelength too short
 - Shear deflection
 - Mass rotational inertia
 - Offset mass and inertia of teeth
 - Extension/compression of neutral axis

Timoshenko formulation vs. Hoppe “Thin Ring”

$$\omega_n = \frac{n(n^2 - 1)}{\sqrt{n^2 + 1}} \sqrt{\frac{EI}{\rho AR^4}}$$

$$\omega_n = n^2 \sqrt{\frac{EI}{\rho AR^4}} \left[1 - \frac{n^2}{2} \frac{I}{AR^2} \left(1 + \frac{E}{k'G} \right) \right]$$

And it got us closer, but...

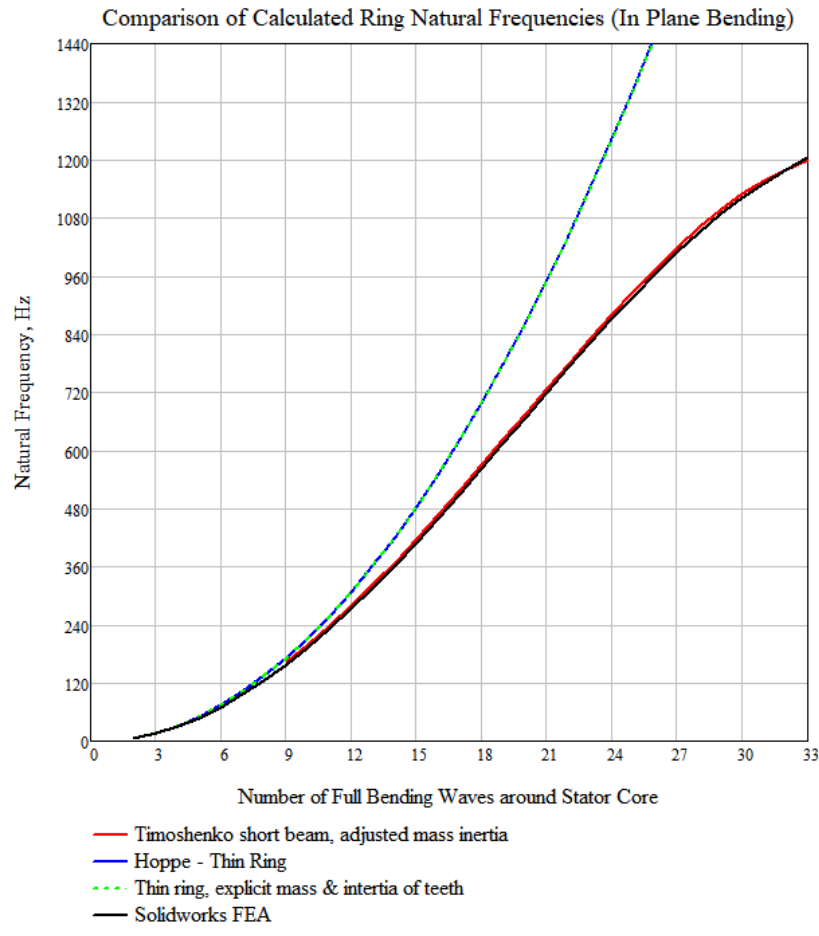


- Timoshenko formulation – uniform cross section
- Actual – slots cause offset center of mass/inertia
- Nodal positions move closer to neutral axis as wavelength decreases

EME Modification:

$$\omega_n = n^2 \sqrt{\frac{EI}{\rho A R^4}} \left[1 - \frac{n^2}{2} \frac{I}{A R^2} \left(\frac{56}{n} + \frac{E}{k' G} \right) \right]$$

Comparison of Results



- Close correspondence to FEA
- Close correspondence to known resonance cases
- Physically defensible if not mathematically rigorous
- Investigation continuing



Thank you for your time and attention!

Thomas J Cunningham, P. E.

EME Associates, Inc.

150 Seco Road

Monroeville, PA 15146

412-351-0290

www.emeassociates.com

tcunningham@emeassociates.com

For further info, come see us at Booth #1149